

Operating temperature	130 °C
Operating Pressure	6.9 E+05 N/m <sup>2</sup>
Line size	12" (0.3239 m)
Line thickness	STD (0.00953 m)
Fluid	Steam condensate (Bulk modulus, K = 2.15806E+09 N/m <sup>2</sup> )
Fluid velocity (V)	3.43 m/s
Fluid density	930 kg/m <sup>3</sup>
Pipe material	A106-Gr.B (Modulus of elasticity, E = 2.07E+11 N/m <sup>2</sup> & Density =7833 kg/m <sup>3</sup> )

## APPROACH -2

"c" is calculated as below

$$c = \sqrt{\frac{K}{\rho + \rho \left( \frac{K}{E_{ml}} \right) \left( \frac{D_{int}}{T} \right)}}$$

$$= \sqrt{\frac{2.15806 \text{ E}+09}{930 + 930 \left( \frac{2.15806 \text{ E}+09}{2.07\text{E}+11} \right) \left( \frac{0.30484}{0.00953} \right)}}$$

$$= 1319.15 \text{ m/s}$$

$$\text{Calculation of pressure rise } (\Delta P) = \rho * c * \Delta V$$

$$= 930 * 1319.15 * 3.43$$

$$= 4.2 \text{ E}+06 \text{ N/m}^2$$

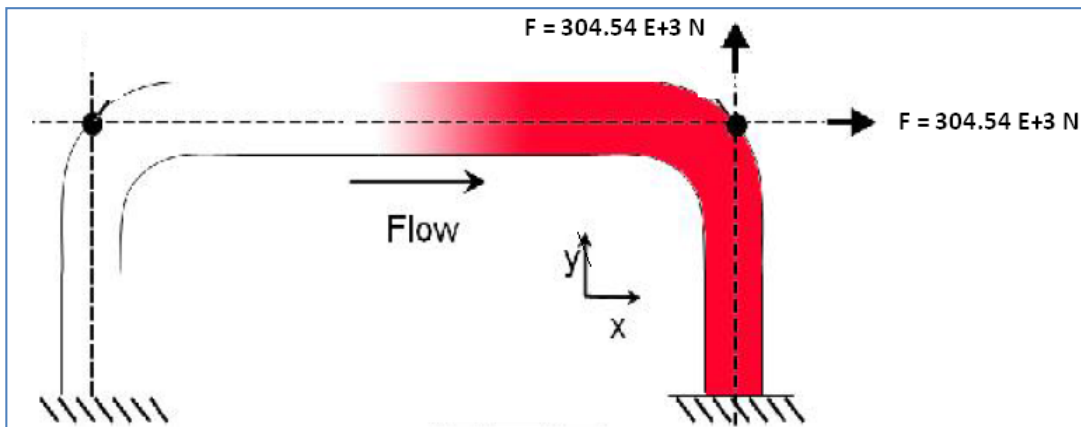
Where "ΔV" is Change in velocity = 3.43 m/s (Fluid flow to stop equal to Fluid velocity)

$$\text{Calculation of unbalanced water hammer force (F) = } \Delta P \times (\text{Flow area})$$

$$= \Delta P * \left[ \frac{\pi}{4} D_{int}^2 \right]$$

$$= 4.2 \text{ E}+06 * \left[ \frac{\pi}{4} 0.30484^2 \right]$$

$$= 304.54 \text{ E}+03 \text{ N}$$



**Considered both positive and negative impulse loads.**

## APPROACH -1

Total Pressure =  $P_{\text{Steady State}} + P_{\text{Transient}}(\Delta P)$

$$= 6.9\text{E}+5 + 4.2\text{E}+6$$

$$= 4.59\text{E}+6 \text{ N/m}^2$$

Calculation of Total force acts on the elbow =  $P * A$

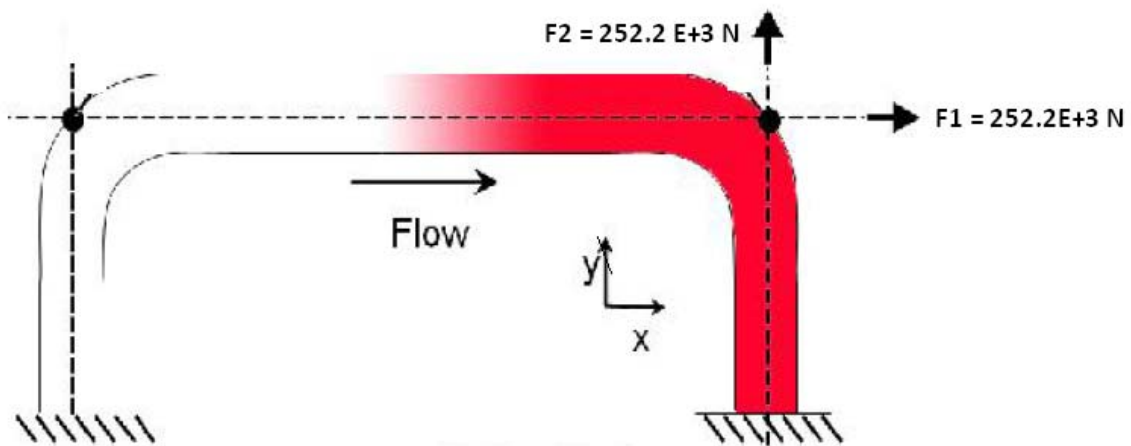
$$= 4.59\text{E}+6 * \pi() * (0.3048^2) / 4$$

$$= 356.8\text{E}+3 \text{ N}$$

Considering 90 deg Bend, Resolving force into two directions

$$F_1 = \sin 45 * 356.8\text{E}+3 = 0.707 * 356.8\text{E}+3 = 252.25\text{E}+3 \text{ N}$$

$$F_2 = \cos 45 * 356.8\text{E}+3 = 0.707 * 356.8\text{E}+3 = 252.25\text{E}+3 \text{ N}$$



**Difference In forces from approach 1 & 2 =>  $304.54\text{E}+3 - 252.2\text{E}+3 = 52340 \text{ N}$**